



Online Ethics Center
FOR ENGINEERING AND SCIENCE

The Case of the Lakewood "Hi-Stak"

Failure of a Stacking Hoist

Author(s)

Anonymous

Year

1995

Description

A summary of a historical case of a heavy equipment accident, which raises questions about responsibility, safety, and design. The case itself includes detailed diagrams and numerical problems.

Abstract

On May 23rd, 1994, West Michaels, an employee of the Buchanan Company, a large construction vehicle manufacturer, had an accident and was seriously injured while operating a "Hi-Stak" crane. The crane had been purchased from the Lakewood Company, a large manufacturing firm, which had designed, manufactured, and sold hundreds self-contained storage/retrieval systems.

Lakewood agreed to investigate the incident and modify the model of the crane to ensure its safety. The case raises questions as to whether Buchanan should give approval to the modified Lakewood design. Another issue that is obviously involved is Lakewood's responsibility to ensure the safety of their products.

The materials include diagrams of the equipment involved, a detailed description of the accident investigation, and several numerical problems based on the case.

accident report, he was thrown about 20 feet. His next recollection was awakening in the hospital bed.

The accident caused the crane wheels on the back side of the pallet to be pulled free of the lower flange of the A588 steel W6X12 girder on the left side. (See Figures 2 and 3).

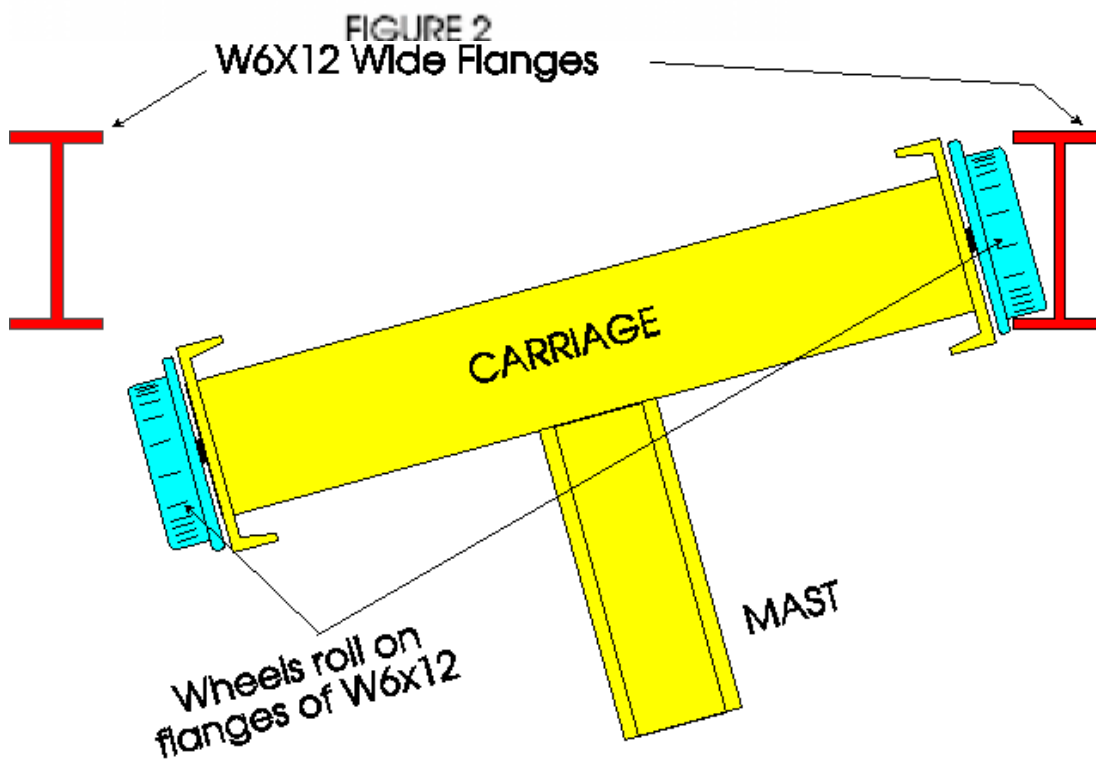
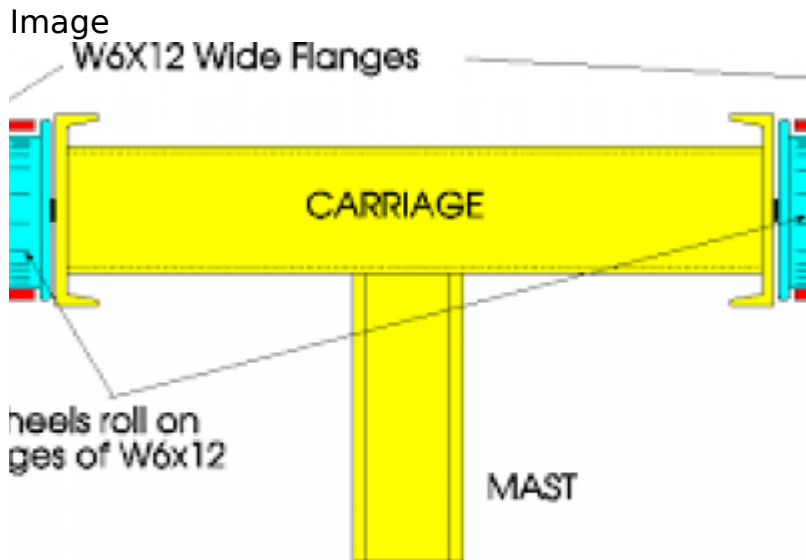


FIGURE 3

Accident Report

Interoffice Memorandum
Buchanan Dies, Inc.
Facility: EAST PEORIA
Date June 22, 1994
Department: Plant Engineering - KK- 1

Attention: Dick Berger
From: Ted Jackson
cc: T. K. Essington, KK-I
Ken Towles, HH Facilities Services
Dave Monroe, Purchasing, A1
Robert Hennelly, Meg., TI

Subject: ACCIDENT INVESTIGATION - Building HH:20E

A. Accident Description:

A Buchanan employee, West Michaels, was injured on May 23rd, 1994 while performing his duties as a FMS operator on the HH Track Shoe line. He was using a Lakewood "Hi-Stak" self-contained storage/retrieval system to store fixtures and dies. The system consists of a top running crane supported by storage racks on each side. The trolley/stacker unit fell to the floor throwing Mr. Michaels about 20 feet. The crane was loaded with a pallet and die weighing a total of 1460 pounds which is well below the system rating of 2000 pounds.

B. Summary & Conclusions:

The employee was doing his job using the tools and procedures provided to him by Buchanan. Buchanan had purchased the unit as a self-contained system from Lakewood, Inc. through their representative Fogerty Brothers Company. The 2 bridge crane girders were obviously too weak to keep the trolley unit in its proper position during normal loading. Our structural calculations, based on the known live and dead loads without adding impact, show that the one girder was overloaded. The girder was not stable under load and deflected laterally enough to allow the trolley wheels to slip off the girder flange. The crane girders must be reinforced for vertical load. The lateral loading caused by the trolley wheel position on the crane girder bottom flange should be reduced or the girder must also be reinforced for lateral stability. Safety lugs mounted on the trolley unit and over the crane girders should also be installed. Lakewood will design the changes and obtain our

agreement on the design. Buchanan HH Facilities Services personnel will make the changes at the direction of Lakewood's serviceman.

We will also work with Stans-Vixen to verify the safety of their similar type storage units.

C. Detailed Investigation & Findings:

The following is a complete history of all my contacts with Lakewood and others that I have received or made since this accident occurred. I have also listed other activities completed during this time period.

Wednesday, May 23rd

1. On Wednesday, May 23, 1994 at 8:47 a.m. a Buchanan employee, West Michaels, was injured while using a Lakewood "Hi-Stak" storage/retrieval system. A copy of the accident report is enclosed.

2. 10 am - I went to HH20E to investigate the accident. The crane trolley had slipped out from between the two crane girders and fell to the shop floor. The load was high above the floor and behind the south girder at time of failure. The evidence at the accident scene indicated that two trolley wheels had slipped off the south girder first. The weight of the load caused the trolley mast unit to pivot about the front girder. The mast swung up striking Mr. Michaels and throwing him about 20 feet north. Fortunately, he was thrown clear of the falling equipment. See also my notes on physical dimensions taken of crane unit, and photographs taken at the scene of the accident.

3. 11:40 am - Dennis Pond reported that the die that fell actually weighed 1285 pounds.

4. 12:32 pm - Ron Fox left a message that the die pallet weighed 175 pounds. Therefore, actual total weight of die and pallet of 1460 pounds is much less than the rated system load of 2000 pounds.

5. A drawing file search found 3 drawings on this "purchased finished" Lakewood system as follows:

- PE15000-L8 Die Storage Loads Layout
- PE15000-L12 Area Layout including Die Storage Area
- PE15000-S26 Lakewood proposal drawing. FMA-22393

These drawings were used to install the storage system and do not include any details on the components of the system.

6. 12:47 pm - Dennis Pond expressed a concern about the rack structure also moving when the stacker crane travels.

7. 1:53 pm - At his request, I updated Larry Lindsey of Y3 Safety on the accident. He was interested in the proper technical terminology for the crane storage system.

8. Purchasing notified Tom Damman at Fogerty Brothers Company of the accident. Fogerty sold Buchanan the Lakewood system in 1986. It has been used successfully and without incident until now. Tom made arrangements to get Lakewood people in as soon as possible. Dave Monroe, Tom Damman and I visited the accident site in the afternoon.

9. 7 pm - I inspected 3 similar "Stans-Vixen" units in building KK. See enclosed data sheet for summary units at buildings KK31B, KK37A and KK48A.

Thursday, May 24th

10. 7:57 am - Dave Monroe called to say Lakewood will be here today.

11. 9 am - Lakewood representatives Nick Magotta (District Sales Mar.) and C. R. (Dick) Johnston (Customer Service Rep.) and Mark Witt from Fogerty Brothers visited the accident site with Dave Monroe and myself.

We met in a conference room and agreed to the following plan of action:

- Lakewood will proceed to repair the trolley unit.
- Lakewood will investigate installing safety lugs or beams on the trolley unit to prevent a similar accident in the future.
- Lakewood will check the need to reinforce or rework the bridge crane girders.
- Lakewood to verify if this system was a standard unit or a specially designed unit for Buchanan's needs. (Sue later found that this 8 foot 3 inch wide aisle unit was one of Lakewood's standard units.) Lakewood to inform us of the timing (schedule) and who will do the actual repair work. Buchanan to write a B94 account work order to accumulate our costs for possible back charging to Lakewood at a future date. Lakewood said they were no longer in this storage unit business and had sold the division to MBT Systems. They said it would take a few days to locate the original drawings for this system. We also agreed that

all contacts must be coordinated through Dave Monroe in Buchanan purchasing or his designated person.

12. Phil Burroughs of HH provided a sketch of the trolley wheel dimensions. The critical distance from the small guide roller to the edge of the trolley wheel tread on the crane girder is only 1 and 5/16 inches. The tread radius reduces the wheel tread contact width to 1 and 5/32 inches. See the sketch enclosed.

13. Dave Monroe telephoned Gene Kent of Crank Midwest Co. to express our concern about the safety of similar Stans-Vixen units he has sold to Buchanan. Dave and I wanted Gene to make arrangements with Stans Vixen people to visit our plant and discuss the safety of their product.

14. 1:20 pm - I expressed our concern about the safety of Stans-Vixen units to Vim Joliat, Technical Sales Mgr. at Allentown, PA based our accident yesterday.

Friday, May 25th

15. 12:20 pm - Ken German called and I updated him on Lakewood's visit.

16. 2:10 pm - Ed Smith from GeneralBusiness Services, Inc. in Springfield called to make an appointment to see the accident site in HH. He said he represented Lakewood as a "claims adjustment" service. He also said he was trying to reach Ken German to set up the visit. I told Ed I would forward his request to Ken German.

17. 2:32 pm - Left message for Ken German about Ed Smith's request to visit accident site on Tuesday May 29th.

Monday, May 28th - Memorial Day Holiday

Tuesday, May 29th

18. 10:30 am - Ed Smith here and visited accident site with Ken German after Ken received advice from Tom Skowronski in Legal, G.O. to allow the visit. A set of 18 photographs were taken at the site and forwarded to Ed Smith via Legal.

19. 10:40 am - Vim Joliat from Stans-Vixen called and set up a conference call with me and his people. He had Jack Remington and Bill Deveny in the room with him. We discussed Buchanan's concern with their crane system design based on the accident we had on May 23rd. They said there were many of these units in use worldwide. They had experienced no similar failures with their cranes. We discussed their

structural design of the trolley and crane units and they could see no problem. They mentioned a maximum trolley wheel load of 3133 pounds on their 2000# rated load system. I expressed a concern about the torsion load on their MC 6x12 ship channel girders due to the eccentric wheel loading condition. They tried to convince me that their system design was good and I said we would probably be calling them again for more information.

20. 11:29 am - Called Stan Hufford and arranged meeting to get information on the Lakewood system from his file on purchase of the system in 1986-87.

21. 2:04 pm - Stan Hufford called and we set meeting for today. Met with Stan at his office and copied several documents about the system purchase and obtained his copy of the 1985 Lakewood system catalog. This system was ordered from Fogerty Brothers on 12-08-86 on P.O. PYXD93375 U.

22. 3:20 pm and 3:52 pm - Discussed the accident and progress with Lakewood, first with Ken German and then with Dave Monroe.

Wednesday, May 30th

23. 9:53 am - Gene Kent of Crank Midwest Co. (Stans-Vixen rep.) called and left message he wanted to see me today.

24. Gene Kent met with me and gave me a list of 38 Stans-Vixen storage units sold to Buchanan. These units were sold to our East Peoria plant, York plant, Aurora plant, and Juliet plant. His list shows 29 units sold to East Peoria plant. He also included two catalogs on the units.

Thursday, May 31st

25. I went to HH:20E and recorded additional dimensions needed to do structural calculations on the Lakewood system. The rack system and aisle width were verified with catalog data as originally ordered.

26. 3:22 pm - Dave Monroe called to inform me that Dick Johnson at Lakewood was having problems locating the storage unit drawings. He may have to travel to Michigan on Monday to find the drawings at MBT Systems.

Friday, June 1st

27. 12:35 pm - Dave Monroe called to say Jim McLeod would be my Purchasing contact next week since Dave would be in Europe on business. Saturday, June 2nd

28. I dropped off to Ken German at Y3 a list of items discussed with Lakewood at the 5-24-94 meeting and other storage system information I had recently gathered from Stan Hufford.

Monday, June 4th (I was sick & off work)

29. 12:59 pm - Angela Kemp from Public Affairs called and left message to set up a safety video taping of the HH accident story for Friday, June 8th at 8 am.

Tuesday, June 5th

30. 7:09 am - Jim McLeod called for update on Lakewood communications.

31. I received by U.S. mail additional data from Gene Kent of Crank Midwest Co. on the Stans-Vixen Stak System. The three pieces of literature covered 1) warranty information, 2) Stak-System installation & operation, and 3) Duff-Norton hoist operating & maintenance instructions.

32. 2:50 pm - Dick Johnson of Lakewood called to say he has found the trolley drawings and they would be sent out today. He also verified that the crane girder section is a W6x12 as we had suspected. Wednesday, June 6th

33. 8:52 am - Jim McLeod called to say Dick Johnson of Lakewood would be here at 10:30 am to visit the accident site again.

Friday, June 8th

34. 8:00 am - JMS did engineering calculations showing the original design of the Lakewood Hi-stack system to be inadequate under the design loads. The major problem is the equivalent 813 pound lateral load on the W6x12 wide flange, which bends the section about its weak axis. The wide flange is overstressed by 3.29 to 1.

Ethical Issues in This Case

1) Should Buchanan give approval to the modified Lakewood design?

Numerical Problems

Numerical Problem #1:

Using the AISC LRFD code specifications applicable to crane rails, determine the design loads to which the rails could be subjected, and determine if the beam is acceptable by AISC standards. The beams were 12 foot long W6x12 made of A588 steel, and you may make worst case assumptions about direction and location of loading. You may also assume that the ends of the beams are pinned.

Numerical Problem #2:

Determine the required lateral deflection of the wide flange assuming that the operator was traveling as slowly as would be required to just barely cause the wheels to slip off of the lower flange.

Numerical Problem #3:

Using energy methods, determine a lower bound on how fast the crane must have been moving to just barely cause the wheels to slip off. Would you consider this design safe to use? Walk this speed. Would you consider this design safe to use?

Numerical Problem #4:

Calculate the maximum possible lateral deflection of the wide flange assuming that the wheels did indeed slip off of the lower flange, and that the operator was running with the crane, failing to stop before hitting the end stops. Note that no permanent plastic deformation of the crane rails was noted after the accident.

Numerical Problem #5:

Using energy methods, determine an upper bound on how fast the crane might have been moving when the wheels slipped off. Does this change your opinion of the safety of the design? Do you feel that it might have been possible that the operator could have caused his own problems by moving too fast with the crane and running into the stops?

Numerical Problem #6:

Shown in figure 3 is a set of calculations provided by Buchanan which they feel proves the original Lakewood design to be inadequate. Do you agree with these calculations, and if not, in what respect?

Numerical Problem #7:

Against Buchanan stated safety procedures, and in direct violation of safety stickers on the crane and listed in the user's manual, the load was transported fully raised, above the operator's head, rather than having been lowered close to the floor immediately after removal from the shelf. Did this operator behavior contribute in any way to the accident and/or to the employee's injuries? Note that the employee was not struck by the falling die, which was on the other side of the crane when it fell. Rather he was involuntarily propelled by the crane.

Numerical (Design) Problem #8:

a) What mechanical measures might you add to the Hi-Stak unit to prevent similar future accidents? Do you feel that the added expense would be worth the added costs? Should Lakewood voluntarily track down and retrofit the hundreds of existing cranes they have sold, realizing that this is the first accident noted since their introduction in 1986?

(b) What electronic interlock controls might you add to the crane system and/or operating controls to prevent the circumstances that contributed to the accident?

Numerical Problem #9:

How fast could a worker travel with the crane and hit the stops if beam safety hooks are used? Assume that both beams will be pulled into complete plastic deformation for this problem. Would you consider this a safe speed? Would you consider the design safe under these conditions?

Numerical (Design) Problem #10:

Redesign the wide flange such that there will be no over-stress under normal loading, assuming that safety devices have been installed to prevent lateral disengagement of the wheels under lateral impact loading.

Lakewood's Side of the Story

Against all safety precautions, Mr. Michaels was transporting the load at full height above the floor, thus directly causing this accident. This is in direct violation of the safety regulations of Buchanan, as well as the safety stickers prominently displayed on the crane, and in the user's manual. Further, Mr. Michaels took the required

safety classes in use of the equipment, where this requirement was emphasized. Loads are never to be transported high, but rather must be lowered to the floor immediately after retrieval from the storage shelves. We contend that this negligence was the full and complete cause of this accident, for the following reasons:

a) Carrying the load level with the upper support beams would directly transmit tremendous impact loads to the support beams, should the operator negligently fail to stop the crane and crash it into the stops. Our tests show that if the loads are transported close to the ground, the crane merely swings forward, reducing the impact loading at the top of the crane to very low levels, greatly reducing the possibility that the wheels can disengage.

b) We admit that should someone crash the crane into the stops, they might actually cause the wheels to come out of the tracks on the back side of the crane. However, should this happen with the load low, the load will merely rotate the crane about 5 degrees, until the pallet is directly under the engaged wheels, or until the load touches the concrete floor. In either case, the rotation of the crane is slight, and rather than propelling the operator across the room, he will probably not even be touched by the crane.

However, should the wheels disengage with the load in high position, the load will rotate the crane 40 to 50 degrees before falling off of the forks. Just like a golf ball, the operator's negligence might cause him to be struck and injured by the rotating crane.

c) Although we have no way of knowing how fast Mr. Michaels ran the crane into the stops, it is apparent to us that it must have been at a significant speed. Statements by Mr. Michaels' co-workers that he has caused several other accidents through carelessness, and testimony of his "hot-dogging" through the plant, confirm this behavior. Further, the fact that several hundred cranes identical to this model have been in operation for 8 years throughout the country demonstrates that the design is safe and reliable if not abused.

d) We have reviewed the calculations made by Buchanan concerning the minimum velocity necessary to disengage the wheels, and would like to point out that this would indeed be a minimum, and has no bearing on how fast Mr. Michaels was actually running at the time of the accident. We would also like to note that although our calculations at this time do appear to show a slight overload condition for the

crane when operated at its rated load of 2000 pounds, the 1460 pound load carried by Mr. Michaels would not over-stress the crane, according to AISC specifications.

Buchanan's Side of the Story

Buchanan Calculations Showing Beam To Be Poorly Designed

Calculations for wheel loading:

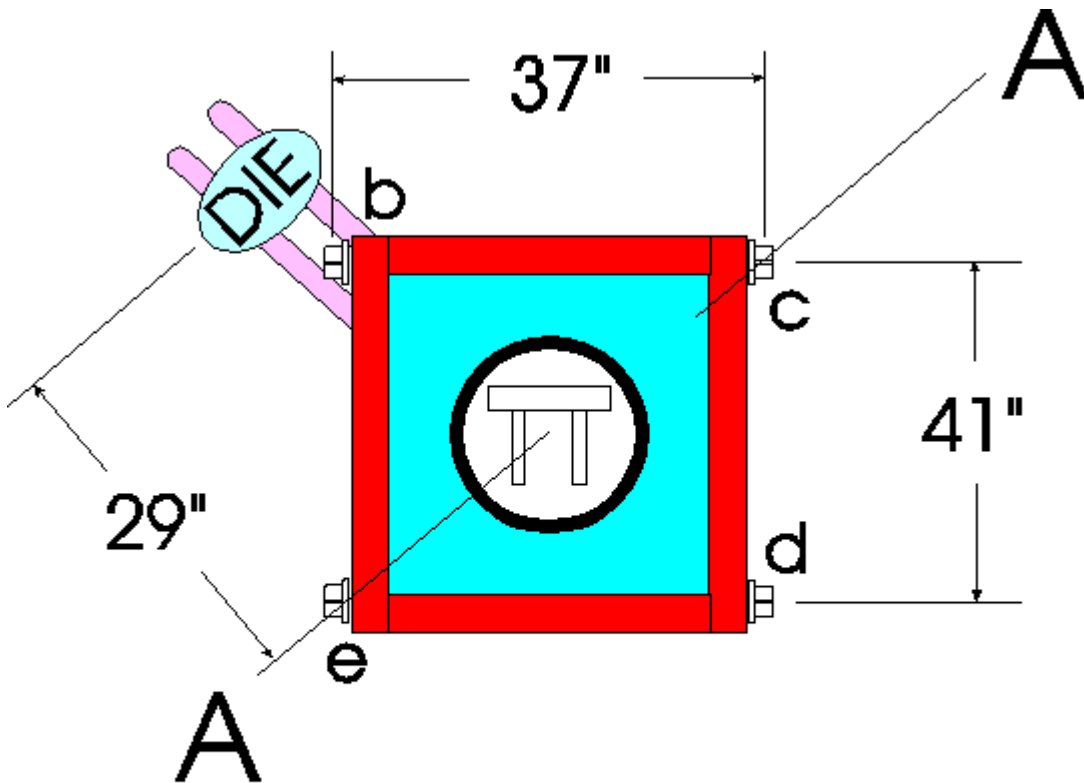


Figure 8

The maximum loading on the W6x12 beam is realized when the pallet and die are rotated directly over one of the carriage wheels, as shown above.

The die and pallet live load rating is 2000 pounds, and can be up to 29 inches from the center of rotation of the crane. The dead load of the crane itself is 1875 pounds, and is located at the point of rotation of the crane. Using AISC-LRFD design procedures, the factored live load, including a 10% allowance for impact (Section A-4.1 and A-4.2) is

$$P_{live} = 2000\# * 1.6 * 1.1 = 3520\#$$

$$\text{The factored dead load} = P_{dead} = 1875\# * 1.2 = 2250\# \text{ (no impact needed)}$$

First, assume that the live die load is applied at the center of rotation. Then the live load on each wheel due to central die load = $3520\# / 4 \text{ wheels} = 880\#$

Since the die load is actually off center, it causes a moment about line A-A:

Moment about line A-A = $3520\# * 29" = 102.080 \text{ in lbs}$, which must be resisted by wheels b and d. Since these wheels are on the hypotenuse of a triangle, they are $[(41/2)^2 + (37/2)^2]^{0.5} = 26.6 \text{ inches}$ from the center of the crane.

Summing moments about wheel line A-A, to determine load on wheel "b":

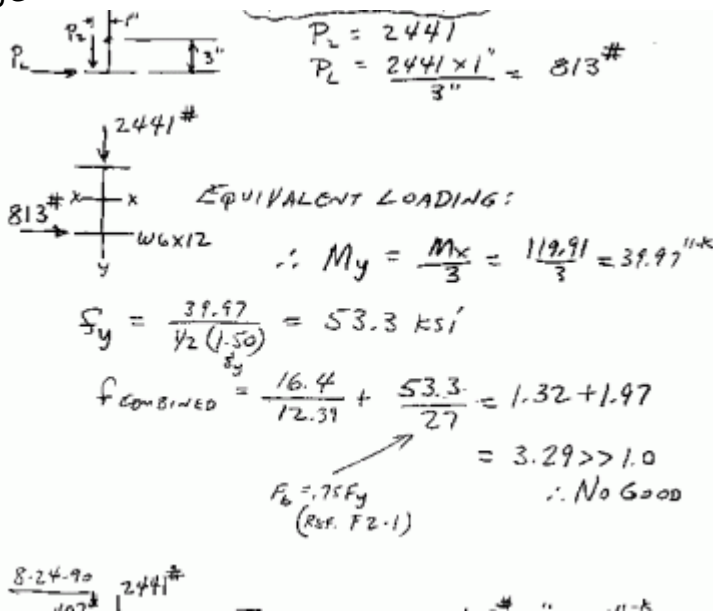
$$\text{Sum moments about line A-A} = 0 = 3520\# * 29" - \text{wheel b load} * 27.6" - \text{wheel d load} * 27.6"$$

Assuming wheels b and d have the same load due to symmetry, Live load on wheel b = $1850\#$.

The load calculations for equivalent beam loading:

P2 is the maximum calculated wheel load, which is 1" off-center. PL is the equivalent lateral load which gives the same torsional twist to the beam as did P2, when P2 is moved to a direct shear position. This lateral load causes significant bending about the weak axis of the beam, as shown in the calculations below.

Image



Notes

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